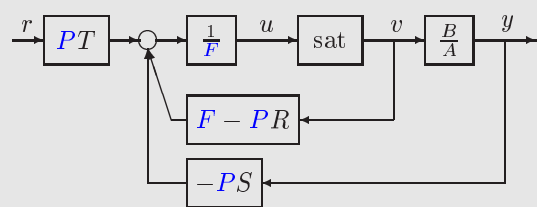




Abstract We consider a linear process, $y = \frac{B}{A}\text{sat}[u]$, controlled by a feedback $Fu = (F - PR)\text{sat}[u] - PSy + PTR$. Here, F, P are *anti-windup compensators*. We choose F as the nominal closed-loop characteristic polynomial $\alpha = RA + SB$ while P is obtained by a \mathcal{H}_2 -optimization, where fast recovery after desaturation is traded against the stability margin by tuning a single parameter, ρ . We show that when tuning to attain a fast recovery after desaturation, this will *also* affect the instant of desaturation in a very desirable way.

System Structure, in polynomial form



- **controller with anti-windup**
 $Fu = (F - PR)\text{sat}[u] - PSy + PTR$
- R, S, T - **nominal controller**
 $Ru = -Sy + Tr$ or
 $u = (1 - R)u - Sy + Tr$
- F, P - **anti-windup compensation polynomials**, see [1]

Tools for Analysis and Design

$$\delta = \text{sat}[u] - u = \frac{P\alpha}{FA}\text{sat}[u] - \frac{PT}{F}r, \quad \alpha = RA + SB$$

$$y = y_i + y_\delta = \mathcal{H}_i r + \mathcal{H}_\delta \delta = \frac{BT}{\alpha}r + \frac{BF}{\alpha P}\delta$$

$$\mathcal{L}_v = \frac{P\alpha}{FA} - 1 \text{ - loop gain around sat}[\cdot]$$

The aim of the design is to obtain a fast recovery after desaturation ($\rho \rightarrow 0$), while preserving stability ($\rho \rightarrow \infty$), see [2]. In case of MIMO systems, see [3].

$$J = \|\mathcal{H}_\delta\|_2^2 + \rho \|(\mathcal{L}_v + 1)^{-1} - 1\|_2^2$$

Minimize J with respect to F and P by selecting

$$F = \alpha, \quad rPP^* = BB^* + \rho AA^*$$

What happens when $\rho \rightarrow \infty$?

$$P \rightarrow A_{stab}, \quad \mathcal{L}_v \rightarrow \frac{A_{stab}}{A} - 1$$

$$\delta \rightarrow \frac{A_{stab}}{A}\text{sat}[u] - \frac{A_{stab}}{A}u_i$$

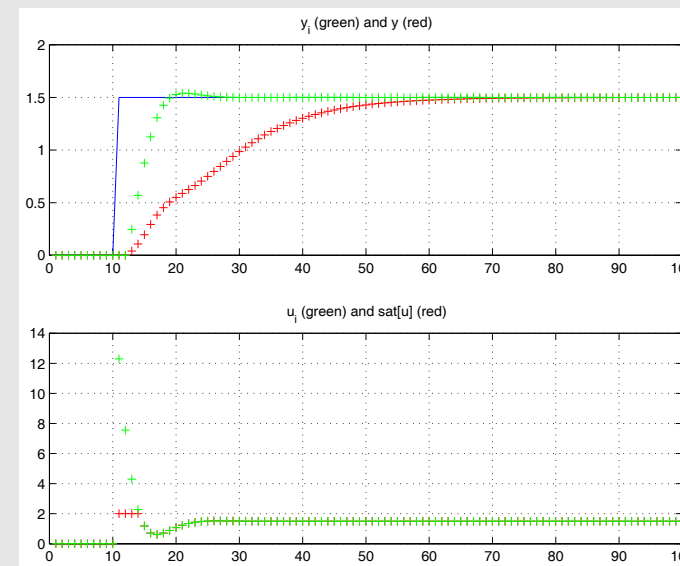
$$\mathcal{H}_\delta \rightarrow \frac{B}{A_{stab}}$$

Assume that A is stable. Then

- $\mathcal{L}_v \rightarrow 0$, which guarantees stability,
- $u = u_i$ which gives $\delta = \text{sat}[u] - u_i$, and
- the plant recovers as $\frac{B}{A}$.

Example: Step response

The control signal, u , saturates ($\text{sat}[u]$ red in the lower diagram) due to the step in r . Since $u = u_i$, (u_i , green in the lower diagram), u desaturates at the same time instant as u_i passes the u^{max} limit (in this case 2). The output recovers with the plant dynamics $\frac{B}{A}$.

What happens when $\rho \rightarrow 0$?

$$P \rightarrow \frac{q^k B_{stab}}{b_0}, \quad \mathcal{L}_v \rightarrow \frac{q^k B_{stab}}{b_0 A} - 1$$

$$\delta \rightarrow \frac{q^k B_{stab}}{b_0 A}\text{sat}[u] - \frac{q^k B_{stab}}{b_0 A}u_i$$

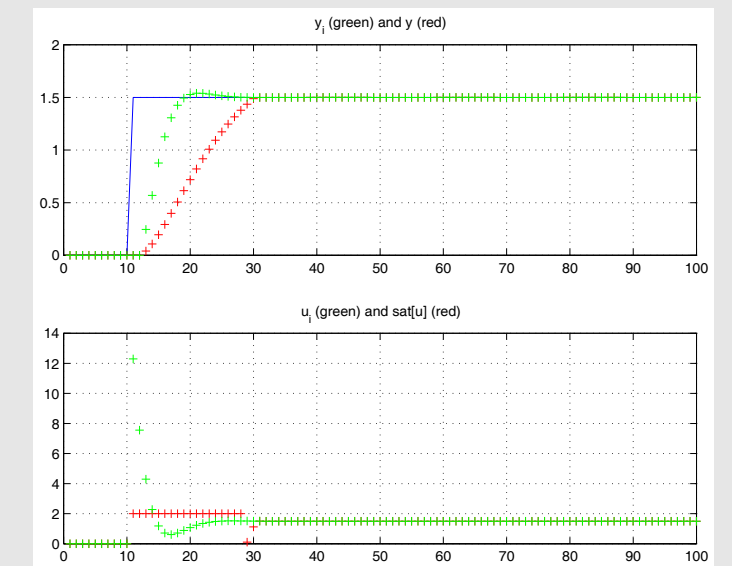
$$\mathcal{H}_\delta \rightarrow \frac{b_0 q^{-k} B}{B_{stab}}$$

Assume that B is "stable". Then

- $\delta \rightarrow \frac{q^k}{b_0}(y - y_i)$, and
- the plant recovers as $b_0 q^{-k}$, (k -time delay in the plant).
- Stability is not guaranteed.

Example: Step response

The control signal, u , saturates ($\text{sat}[u]$ red in the lower diagram) due to the step in r . Since $\delta = \frac{q^k}{b_0}(y - y_i)$, u will desaturate k samples before the intersection $y^{max} = y_i$ and y recovers completely within k samples after desaturation (in this case $k = 2$).



Ideal Phase

$$y = y_i$$

$$u = u_i$$

$$Ru = -Sy + Tr$$

Phase of Saturation

What about stability?
Study $\mathcal{L}_v = \frac{P}{A} - 1$!

How does saturation affect the plant output?
Study $y_\delta = \frac{B}{P}\delta$!

When does the actuator desaturate?
Study $\delta = \frac{P}{A}u_{min}^{max} - \frac{P}{A}u_i$!

Phase of Recovery

How does the plant recover?
Study $\mathcal{H}_\delta = \frac{B}{P}$!

Conclutions

Tuning $\rho \rightarrow 0$ gives the most desirable performance, as long as stability is maintained, and repeated re-saturations, which might occur after desaturation, are few. These undesired effects can be predicted by study a of \mathcal{L}_v in a Nyquist plot.

References

- [1] S. Rönnbäck, *Linear Control of Systems with Actuator Constraints*. PhD thesis, Division of Automatic Control, Luleå University of Technology, Sweden, 1993.
- [2] M. Sternad and S. Rönnbäck, "A Frequency domain approach to anti-windup compensator design," Inst. Technol., Uppsala Univ., Sweden, Rep. UPTEC 93024R, 1993.
- [3] J. Öhr, Mikael Sternad and Anders Ahle'n *Anti-Windup Compensators for Multivariable Systems*. European Control Conference, Brussels, Belgium, 1997, vol. 2, pp. FR-A G5.